

Bending Stiffness Collapse, Buckling, Topological Bands of Freestanding Twisted Bilayer Graphene

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The freestanding twisted bilayer graphene (TBG) is unstable, below a critical twist angle $\theta_c \sim 3.7^\circ$, against a moiré (2×1) buckling distortion at $T = 0$. Realistic simulations reveal the concurrent unexpected collapse of the bending rigidity, an unrelated macroscopic mechanical parameter. An analytical model connects bending and buckling anomalies at $T = 0$, but as temperature rises the former fades, while buckling persists further. The (2×1) electronic properties are also surprising. The magic twist angle narrow bands, now eight in number, fail to show zone boundary splittings despite the new periodicity. Symmetry shows how this is dictated by an effective single valley physics. These structural, critical, and electronic predictions promise to make the freestanding state of TBG especially interesting.

Two-dimensional (2D) bilayers and multilayers with variable lattice mismatch and/or twist angles exhibit a host of physical properties that also hold promise for applications [1–4]. With exceptional electronic properties at the magic twist angle, twisted bilayers graphene (TBG) are prominent among them [5–7]. Experimental bilayers are generally studied in deposited/encapsulated configurations, which preserve a flat geometry and the twist-related moiré pattern plays no mechanical role. Yet, TBG may also be realized as freestanding [8, 9]. Below a critical twist angle θ_c , moiré related structural instabilities and a variety of “buckled” states were suggested by pioneering freestanding simulations [10–12], but the actual nature and properties of the true TBG buckled state remain unknown. We use here theory and simulation to show that the moiré buckled state formed at low twist angle is quite different from expectations. Mechanically, it is accompanied by the unanticipated collapse of the TBG macroscopic bending rigidity. Electronically, the magic twist angle narrow bands, now doubled in number, are unexpectedly degenerate at zone boundaries, the vanishing Bragg scattering symmetry motivated reflecting single valley physics.

Starting with molecular dynamics (MD) simulations, large size model TBGs with variable twist θ and variable numbers $N_{\text{moiré}}$ of moiré cells were constructed with periodic boundary conditions in the (x, y) plane. Based on well-tested interatomic interactions and protocols (detailed in SI.I) we sought the zero stress equilibrium $T=0$ structure versus θ . We found that, similar but not identical to suggestions [10–12], two regimes emerge, separated by a structural phase transition at a critical $\theta_c \approx 3.77^\circ$. Above θ_c the two layers remain flat and specular relative to the central plane (Fig. 1b). Below θ_c the layers jointly buckle giving rise to a “moiré (2×1)” cell doubling along armchair direction x , leading to two inequivalent, z -antisymmetrical AA nodes, AA1 (up) and AA2

(down) per cell, as in Fig. 1a. The magnitude of buckling is large. At the magic twist angle $\theta_m \approx 1.08^\circ$ for example the zigzag z -corrugation is $\approx 10\text{\AA}$ (Fig. 1a,c). A competing ($\sqrt{3} \times \sqrt{3}$) buckling distortion, with one AA1 (100 % up) hexagonally surrounded by AA2 and AA3 (50 % down), was also found. It led to a slightly lower energy gain, and its details are not further pursued here (see however SI.VIII).

The energy gain driving the buckling distortion at $\theta < \theta_c$ is interlayer, with increased AB and BA, Bernal stacked areas, relative to the flat, unbuckled state’s. That gain is balanced by an intralayer cost concentrated at the AA nodes, now transformed into buckling “hinges” AA1 (up) and AA2 (down). As θ decreases, the 2D density of AA nodes, thus of hinges, drops $\sim \theta^2$, and so does the cost, eventually favoring buckling for $\theta \leq \theta_c$. The transition, simulated by maintaining zero external stress and zero temperature, was found to be continuous. We define the $T = 0$ K buckling order parameter Q_0 as the large-size average Fourier component of the (2×1) moiré corrugation

$$Q_0 = \frac{a_{\text{Gr}}}{N_x N_y A} \left\langle \sum_{n=1}^{N_{\text{at}}} z_n \exp \left(-\frac{2\pi i}{l_x} x_n \right) \right\rangle \quad (1)$$

where a_{Gr} is graphene’s lattice constant, $l_x = \sqrt{3}\lambda$ is the size of the buckled unit cell along the armchair direction x (Fig. 1a,b), $\lambda \sim a_{\text{Gr}}/\theta$ is the moiré lattice constant (Fig. 1a,b), x_n and z_n are coordinates of the n -th atom ($n = 1, 2, \dots, N_{\text{at}}$), $A = \sqrt{3}\lambda^2$ is the buckled unit cell area, and N_x, N_y are the number of cell replicas along x and y . This $Q_0(\theta)$ is proportional to the buckling induced bilayer thickness increase $\langle (z_{\text{AA1}} - z_{\text{AA2}}) \rangle$ (inset of Fig. 1c). Simulation for a set of θ values (see SI.I) characterized by a sufficient numbers of moiré cells $N_{\text{moiré}}$ showed, at $T = 0$, a growth of $Q_0(\theta < \theta_c)$ (Fig. 1c) well approximated by a power-law rise $Q_0(\theta) \approx 0.48\Theta(\theta_c - \theta)(\theta_c - \theta)^\beta$, where

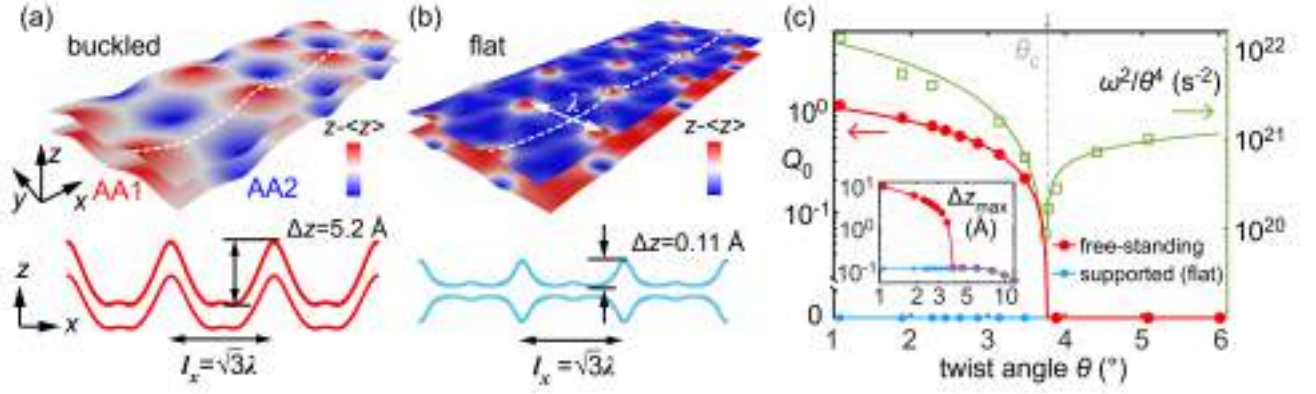


FIG. 1. Visual models of small twist angle TBG structure at $T = 0$. Out-of-plane displacement of (a) optimal free-standing (2×1) buckled and (b) flat structures, magnified by factors 3 and 100 respectively, are sketched for twist angle $\theta_m = 1.08^\circ$, where the moiré size $\lambda \approx a_{Gr}/\theta_m \approx 13$ nm. (c) Twist angle dependence of the optimal buckling order parameter Q_0 (red and blue, left axis) for $N_x = N_y = 1$, $T = 0$ K, and of the normalized soft phonon frequencies ω^2/θ^4 (right axis, green). Critical twist angle $\theta_c \approx 3.77^\circ$ marked by grey dashed line. Red and green lines are power law fits as described in text. Inset shows the maximum out-of-plane corrugation Δz_{\max} .

Θ is Heaviside's function, and $\beta = 0.7(0)$ a critical exponent. Within fitting uncertainty, reflected by the second decimal in brackets, this exponent differs from $1/2$, that could be expected for a classical $T = 0$ transition.

As in other displacive phase transitions, the local free energy around equilibrium supports a soft buckling phonon mode ω_i ($i = (+, -)$ refers to above or below θ_c), a mode which will also control critical fluctuations at $T > 0$. Its frequency was extracted from oscillations around equilibrium of a (2×1) moiré cell by starting the dynamics with $Q = Q_0 + \delta Q$, with $|\delta Q|/Q_0 \ll 1$, while maintaining $T = 0$ and zero stress. Because the cell area $A(\theta)$ varies as θ^{-2} , it is convenient to further normalize the soft mode frequencies to constant area in the form ω^2/θ^4 . Power law fits near the singularity at θ_c yield (Fig. 1c), $\omega_i^2/\theta^4 \sim a_i|\theta - \theta_c|^{\gamma_i}$, with $\gamma_+ = 0.3(7)$, $\gamma_- = 1.3(0)$, with $a_+ = 8.58 \times 10^{20} \text{ s}^{-2}$, $a_- = 3.24 \times 10^{21} \text{ s}^{-2}$. Note the strongly asymmetric, again unusual exponents. We observe nevertheless that $\gamma_-/\beta \approx 2$ as in standard mean-field theory.

More interestingly, the buckling amplitude Q_0 and its soft mode frequency are not the only critical quantities at $\theta \rightarrow \theta_c$. We found an unexpected macroscopic partner, the bilayer's bending stiffness. Defined for direction $\mu = (x, y)$ as $D_\mu = dF/d[(\partial^2 h/\partial \mu^2)^2]$, where F is the Helmholtz free energy density, $\partial^2 h/\partial \mu^2$ is the μ -th component of the 2D Laplacian, and h the bilayer's corrugation profile $h(x, y)$ (Fig. 2). Controlling the membrane's deviations from planarity, D_μ determines the macroscopic flexural mode dispersion along μ , $\omega_\mu(q_\mu) = (D_\mu/\rho_{2D})^{1/2}q_\mu^2$, of an infinite membrane of 2D density ρ_{2D} .

The freestanding TBG bending stiffness D_μ was extracted by starting simulations with a slight x -compression (Fig. 2), i.e., $L_\mu = L_{0\mu} - \delta L$, of the bilayer's

zero-pressure equilibrium size. Either the initial energy increase δE , or the ensuing flexural oscillation ω_μ yield $D_\mu = \lim_{\delta E \rightarrow 0} \frac{L_\mu^4 \delta E}{\pi^4 A h^2} = \frac{\rho_{2D} L_\mu^4 \omega_\mu^2}{16 \pi^4}$ [13]. The resulting D_x is shown in Fig. 2. For $6^\circ < \theta < 30^\circ$, D_x is close to $2D_0$, where $D_0 = 1.44$ eV is the bending stiffness of monolayer graphene [14], the factor 2 reflecting free sliding between the two layers [15, 16]. At the opposite end $\theta = 0^\circ$, layers are instead locked in Bernal's AB stacking. That pushes D_x up to $D_B \approx 100$ eV, now reflecting the large in-plane stiffness of graphene [17, 18]. The novelty is that between these extremes, D_x drops below $2D_0$ when $\theta \lesssim 6^\circ$, critically collapsing at θ_c , and rising immediately below

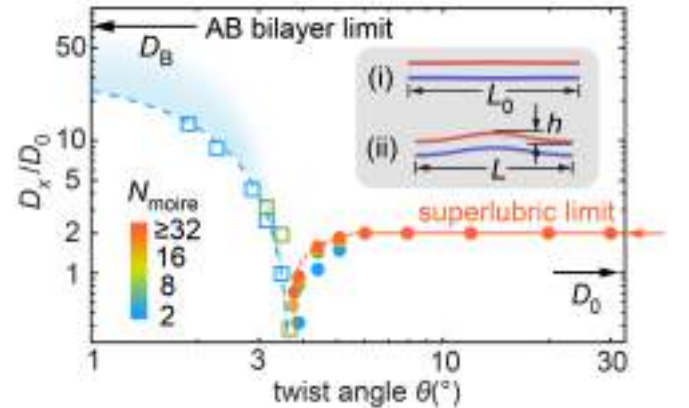


FIG. 2. Freestanding TBG bending stiffness D_x/D_0 , normalized to the monolayer's D_0 , from zero stress simulations at $T = 0$. Note the critical collapse (red circles for $\theta > \theta_c$, blue squares for $\theta < \theta_c$). Dashed and dotted curves are power-law fits near θ_c . Multiplicity of circles at same twist angle shows convergence for simulation cells with increasing size $N_{\text{moiré}}$ (the $N_{\text{moiré}}$ dependence is weak for $\theta < \theta_c$). Inset: simulation protocol (see text).

towards D_B . Near θ_c the collapse is critical

$$D_x(\theta) \sim c_i |\theta - \theta_c|^{\epsilon_i} \quad (2)$$

with exponents $\epsilon_+ = 0.2(2)$ and $\epsilon_- = 1.4(4)$, and $c_+ = 2.4$ eV and $c_- = 7.5$ eV, for $\theta > \theta_c$ and $\theta < \theta_c$ respectively.

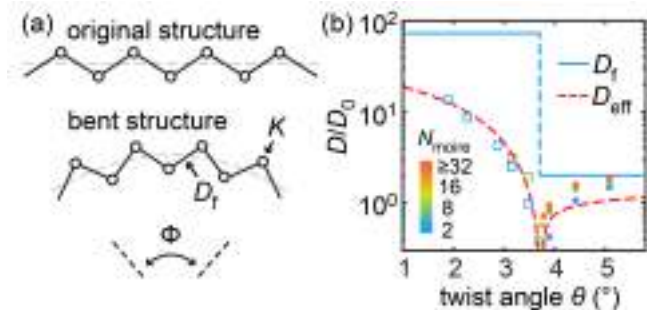


FIG. 3. Zigzag model of bending stiffness D_{eff} in a buckled bilayer. (a) Unbent and bent models; (b) bending stiffness from real simulations of Fig. 2 (circles and squares) and from zigzag model, Eq. (3) (red dashed line). Assumed flat piece stiffness D_f (blue line). (more detail in SI. V).

Why does a *macroscopic* mechanical parameter like $D_x(\theta)$ drop critically at the *microscopic* buckling transition? We developed an analytical “zigzag” model that explains it. As sketched in Fig. 3a the buckled structure can roughly be modeled as a zigzag shape where flat (AB-commensurate) regions are separated by maximally bent (AA-centered) hinges. The total length of the system along the buckling direction is $L_x = N_x l_x$. Upon bending along x both hinges and flats undergo deformation, and the free energy increase with bending angle Φ is $F(\Phi) = \frac{D_f K l_y}{2N(2D_f l_y + K l_x)} \Phi^2$ where D_f is the bending stiffness of the flat pieces, K the angular stiffness of the hinges, and l_y the bilayer size perpendicular to the bending direction x . Defining an effective bending stiffness $F = \frac{D_{\text{eff}} l_y}{2L_x} \Phi^2$, one obtains, using $l_y = \lambda$,

$$D_{\text{eff}} = \frac{D_f}{1 + 2D_f/\sqrt{3}K}. \quad (3)$$

We can now assume $D_f = D_B \approx 100$ eV of the flat pieces for $\theta < \theta_c$, dropping to $D_f = 2D_0 = 2.88$ eV for $\theta > \theta_c$ (Fig. 3b). The collapse of $D_x(\theta)$ at θ_c is controlled by that of the hinge stiffness K , connected to the buckling frequency ω_{\pm} by simple mechanics

$$K \propto \rho_{2D} l_x^4 \omega_{\pm}^2. \quad (4)$$

Inserting ω_{\pm} into Fig. 3b, theoretical and simulated bending stiffnesses agree fairly well, both in magnitude and in critical scaling (details in SI.V). Thus the TBG bending stiffness collapse is a direct consequence of that of the buckling modes ω_{\pm} . In return, the buckling criticality must be influenced by the bending one. The coexistence of these two coupled degrees of freedom,

with important cross correlations, is likely to account for the unusual exponents.

We come next to two important properties predicted for the freestanding TBG buckled state, namely temperature behaviour, and the narrow band electronic structure at the magic twist angles.

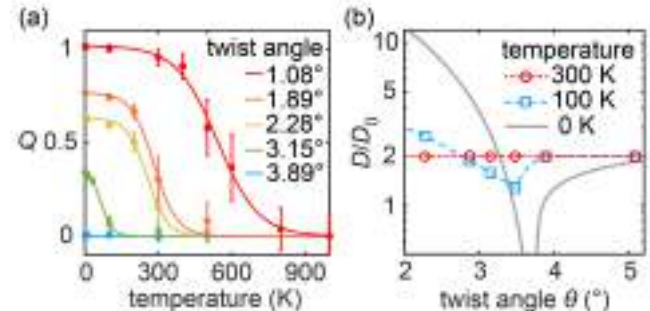


FIG. 4. Effect of temperature. (a) Buckling order parameter at high temperatures. The expected critical behaviour is smoothed by small simulation size (three (2×1) cells at each θ). (b) Bending stiffness at $T = 0$ (gray, from Fig. 2), 100 K (blue) and 300 K (red). Note the extreme sensitivity to temperature.

Temperature. Finite temperature MD simulations show that at small twist angle the buckling persists up to high temperature. Flexural fluctuations, abundant and not gapped, do not cancel the buckling order parameter, which survives up to a remarkable ≈ 500 K at the magic twist angle (Fig. 4a). If bending could be ignored, this robust buckling order should drop at some T_c with 3-state Potts universality, whose behaviour is critical as opposed to first order, despite the presence of Landau 3rd order invariants [19] [20]. Unfortunately, size limitations obscure the high temperature behaviour, replacing it with the smooth crossovers of Fig. 4a (see SI. IV), equally compatible with either continuous or discontinuous decays. Contrary to the buckling robustness, bending stiffness changes dramatically with temperature. The singularity near θ_c is quickly wiped out (Fig. 4b), the two layers eventually bending independently despite the large buckling. Reflecting that, the flexural fluctuations of a TBG near θ_c should grow anomalously at low temperature, when D_x is small, but not above.

Electronic structure. Near the magic twist angle $\theta_m = 1.08^\circ$, the flat TBG has four ultra-narrow bands, whose physics has been at the center of much attention [5, 6]. What will happen of these bands in the freestanding TBG, where the buckling sets in? We should anticipate eight narrow bands, separated by zone boundary gaps caused by the large (2×1) distortion. We carried out tight-binding calculations at θ_m and compared buckled and unbuckled TBG. Shown in Fig. 5 [21] are the 8 bands

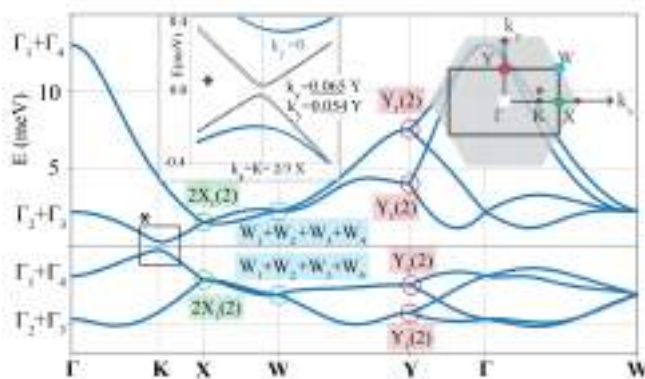


FIG. 5. The eight narrow bands of the (2×1) buckled TBG at the twist angle $\theta = 1.08^\circ$ (see note [21]). Inset * shows how the gap closing point is now split into two, above and below (not shown) the point K where it lies in the flat TBG.

of the buckled state. The bands, almost a factor 2 wider, display important novelties. First, unlike the unbuckled TBG (or even the non-optimal (1×1) buckled state [12]) the Dirac zero gap, formerly at point K , is now split into two close-by points $K \pm k_y$ (see inset). Second, and striking, zone boundary splittings at W and X points – expected because (2×1) buckling removes the C_{3z} symmetry, and Bragg scattering should in principle take place – do not occur. This anomaly calls for a full symmetry analysis.

The buckled structure has, unlike the flat one, a non-symmorphic space group $P2_12_12$ (no. 18) that includes $\{1 | 0\}$, $\{2_{010} | 0, 1/2, 0\}$, $\{2_{100} | 1/2, 0, 0\}$ and $\{2_{001} | 1/2, 1/2, 0\}$, (in Seitz notation, with fractional translations) which we shortly denote as E , C_{2y} , C_{2x} and C_{2z} , respectively. The Bloch states of the 8 bands at the high symmetry points transform as the irreducible representations (irreps) of the corresponding little groups (see Table S3 in SI). In our reference frame only C_{2y} commutes with the generator of $U(1)$ valley symmetry, while C_{2x} , and thus C_{2z} , exchange the valley index – as time-reversal symmetry \hat{T} also does. It follows that, e.g., at the Γ -point, C_{2x} and valley $U(1)$ get promoted to $SU(2)$. That enforces degeneracy between eigenstates with opposite parity under C_{2x} and same parity under C_{2y} [22], making Γ_1 degenerate with Γ_4 , and Γ_2 with Γ_3 . The interplay between valley $U(1)$ and the non-symmorphic group's fractional translations has further consequences at zone boundary points W and X . Physical insight is obtained by switching to a single valley representation where valley index is conserved [23]. This representation obeys the magnetic space group $P2_12_12'$ (no. 18.19) generated by E , C_{2y} , $\hat{T}C_{2x}$ and $\hat{T}C_{2z}$. The four flat bands per valley can be generated by Wannier orbitals centered at the Bernal stacked regions. Their centers correspond to the Wyckoff positions $4c$ in the magnetic space group $P2_12_12'$ [24], whose co-representation contains just the

identity and thus allows a single irrep. The single-valley elementary band representation is [24]: $2\Gamma_1(+)\oplus 2\Gamma_2(-)$ and $2Y_1(+)\oplus 2Y_2(-)$, where (\pm) indicate the parity under C_{2y} ; $X_1X_2(2)$ and $W_1W_2(2)$, transforming under C_{2y} , respectively, as σ_3 , the third Pauli matrix, and $i\sigma_3$ (see Table S3). Along the path $X \rightarrow W$, $k_y \in 0 \rightarrow \pi$, the irreducible representation remains twofold degenerate and transforms under C_{2y} like $e^{ik_y/2} \sigma_3$. This now yields double degeneracies within a single valley. Bearing in mind that the two valleys must be further degenerate at all high-symmetry points and paths that are invariant under C_{2x} , one readily recovers the fourfold degeneracy at W . In fact all “accidental” degeneracies of the band structure in Fig. 5 are explained in terms of single valley physics, enforced by non symmorphycity.

These one-electron degeneracies, and the ways they might be broken by interactions, represent an interesting question for freestanding TBG experiments, where topology should also play a role. Since narrow bands do admit an elementary representation, the arguments used in the unbuckled case to diagnose a fragile topology [23] do not strictly apply here. The unexpected similarity of buckled bands to the (2×1) BZ folded unbuckled ones (Fig. S12), nonetheless suggests that the topological properties remain similar. Thus [22, 25] the interplay between Coulomb repulsion and electron-phonon coupling to Kekulé modes should split the degeneracies and open the gaps that are absent at the one electron level, stabilising topological insulators in (2×1) buckled TBG, giving rise to novel fractional fillings absent in the flat state (See SI. IX).

In summary, several important phenomena are predicted to occur once freestanding TBG will be realized. First, a zigzag buckled state should set in with a critical behavior as a function of twist angle $\theta \rightarrow \theta_c \approx 3.7^\circ$. At $\theta \approx 1^\circ$ the normals to the bilayer should deviate from \hat{z} by a sizeable $\sim \pm 3^\circ$ (SI. VII), experimentally observable. Second, the macroscopic bending stiffness, a crucial mechanical parameter for a membrane, should collapse at the buckling transition, giving rise to gigantic flexural fluctuations already at very low temperatures. Third, the buckling distortion should survive up to relatively high temperatures, whereas the bending stiffness anomaly will on the contrary dwindle upon heating. Fourth, narrow electronic bands are predicted for the buckled magic TBG displaying unexpected single-valley degeneracies, to be broken by interactions, with the possibility of doubling the number of quantized fillings upon gating. That should offer a richer playground for topological features and insulating states than for flat TBG. Other properties including kinetic and tribological behaviour will be addressed in follow-up work. Similar buckling phenomena could take place in freestanding bilayers of other 2D materials, now being pursued.

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Supplementary Information

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I. SIMULATION DETAILS

Twisted bilayer graphene (TBG) with periodic boundary conditions (PBC) along x and y directions are constructed for a discrete set of twist angles θ ranging from 1.08° to 30° chosen for supercells of reasonable sizes [1]. For the freestanding systems, no additional constraints are imposed along the out-of-plane (i.e., z) direction. To describe supported systems, z -direction springs with spring constant k_z are tethered to each carbon atoms to mimic the constraints from the substrate. The interlayer and intralayer interaction is described by the registry-dependent Kolmogorov-Crespi potential with local normals [2, 3] and REBO force field respectively [4]. All simulations are performed with open-source code LAMMPS [5, 6].

Structural optimization. During the optimization, the simulation box adaptively changes size, so that the in-plane stress is fixed to zero, $p_{xx} = p_{yy} = 0$. The FIRE algorithm [7] is used to minimize energy during structural optimization (together with conjugate gradient algorithms with several loops to optimize the box size). Minimization stopped when the largest single atom force $|F_i| < 10^{-6}$ eV/Å. Unless otherwise specified, all structural optimization used this convergence criterion.

Bending simulation. A compression protocol [8, 9] is used to extract the bending stiffness of TBG. As shown in Fig. 2 in the main text, by compressing the simulation box (along x direction), the out-of-plane corrugation appears with height h (in bending simulations, $k_z = 0$). The compression strain $\varepsilon = (L_0 - L)/L_0$ in our bending simulations is chosen to be $\varepsilon \lesssim 0.1\%$, thus the ratio between the bending corrugation height ($h \sim L_0\sqrt{\varepsilon}/\pi$) and the length $h/L_0 \ll 1$. Since the free energy of the system increases during the compression-induced bending, the system will spontaneously oscillate after the release of the boundary constraints. From that we obtain the flexural oscillation frequency and thus the bending stiffness. In order to achieve free oscillations, the Nosé-Hoover thermostat and barostat are applied to the whole system during the simulation [10] with $T \rightarrow 0$ K and $p_{xx} = p_{yy} = 0$. The flexural frequencies are checked to be independent of the damping coefficient used.

II. BUCKLING ORDER PARAMETER AT VARIABLE TWIST ANGLE

As in main text, we start at finite temperature with the same order parameter definition Q^* for the (2×1) buckling distortion in the form:

$$Q^* = \frac{a_{\text{Gr}}}{2L_x L_y} \left\langle \left| \sum_{n=1}^N z_n \exp\left(-\frac{2\pi i}{l_x} x_n\right) \right| \right\rangle \quad (\text{S1})$$

where $L_x = N_x l_x$ and $L_y = N_y l_y$ is the size of the box, $l_x \times l_y$ defines the size of one (2×1) moiré cell, N_x and N_y are the number of replicas along x and y directions. For the smallest simulation box ($N_x = N_y = 1$) and at $T = 0$ where the modulus is redundant, Eq. (S1) coincides with Fig.1 in maintext. At nonzero temperatures, two problems arise with this definition of order parameter, a quantity that should represent the Bragg scattering magnitude produced by the symmetry breaking buckling distortion and nothing else. The first problem is that the coordinate difference of two separate TBG layer centers-of-mass exhibits a noticeable (x, y) random walk, permitted by the superlubric nature of their incommensurate contact, and by the finite supercell size. The larger the size, and the lower the temperature, the smaller this artifact. Because it gives rise to complex values of Q , we can eliminate it by taking the complex modulus as is done in Q^* .

The second problem is that temperature causes fluctuations with (2×1) periodicities, along with all other wavelengths, even in absence of distortions. This unwanted additional term had better be subtracted for the order parameter to yield correctly the distortion magnitude, representing the (2×1) Bragg reflection magnitude of a hypothetical scatterer. This correction will be discussed and eliminated in subsequent section III.

From its definition, it is apparent that there is a positive correlation between the order parameter Q_0 and the maximum out-of-plane deformation $\Delta z_{\text{max}} = \max(z) - \min(z)$. For systems at $T = 0$ K, we could use either parameter to represent the out-of-plane buckling magnitude. As shown in Fig. S1, for buckled structures ($\theta < \theta_c$), both Δz_{max} and Q_0 are well fit by power laws, with an exponent $\approx 0.7(0)$. For $\theta > \theta_c$, the order parameters is zero – no buckling.

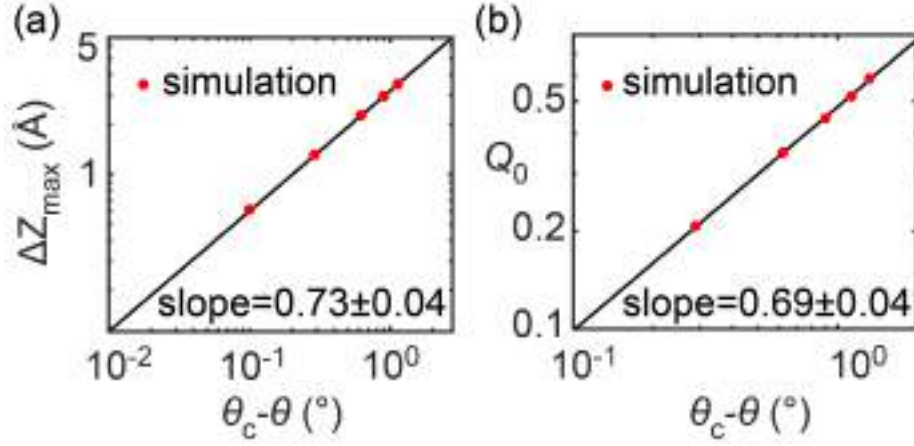


FIG. S1: Fitting of the out-of-plane corrugation Δz_{\max} and order parameters Q_0 of the buckling systems as a function of twist angle θ , at $T = 0$ K. The critical angle is $\theta_c = 3.77^\circ$. Parameters used here are $T = 0$ K, $N_x = 3$ and $N_y = 1$.

III. TEMPERATURE CORRECTED ORDER PARAMETER

As mentioned above, the order parameter definition needs to be corrected in the case of finite temperature.

A straight thermal fluctuation term $R(T)$ with buckling-unrelated (2×1) periodicity is to be subtracted from Q^* :

$$Q = Q^* - R \quad (\text{S2})$$

where R should be the nonzero (2×1) Fourier amplitude due to ordinary thermal fluctuations, if the TBG remained hypothetically unbuckled. As an approximation to that, we extrapolate to $T \leq T_c$ the (2×1) Fourier amplitude evaluated at $T \gtrsim T_c$, where the distortion has disappeared

$$R(T \lesssim T_c) = Q^*(T \gtrsim T_c) \quad (\text{S3})$$

This thermal fluctuation with $k = 2\pi/l_x$ further depends on the size of the bilayer and of course on temperature [11], $R(T) \propto l_x \sqrt{T}$, so that finally

$$R(T) \propto \theta^{-1} \sqrt{T} \quad (\text{S4})$$

The thermal (2×1) background R is extracted at $\theta = 3.89^\circ$, and its temperature dependence, needed for the extrapolation, is further verified at $T > T_c$, where buckling is absent. As shown in Fig. S2, the scaling given by Eq. (S4) is reasonable.

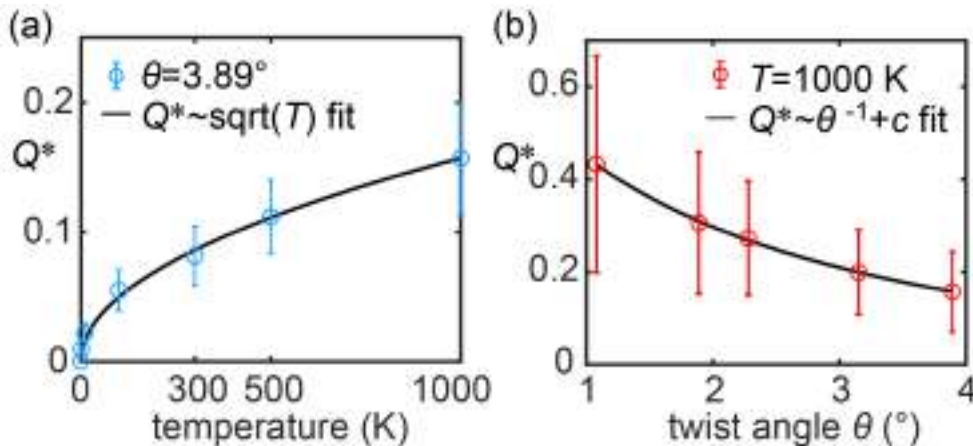


FIG. S2: Verification of Eq. (S4). (a) The simulated order parameter Q^* scales as $T^{1/2}$. (b) The simulated order parameter Q^* scales as θ^{-1} . Except for temperature, parameter used here are same as those in Fig. S1.

A direct comparison of the original Q^* and the correct Q is shown in Fig. S3.

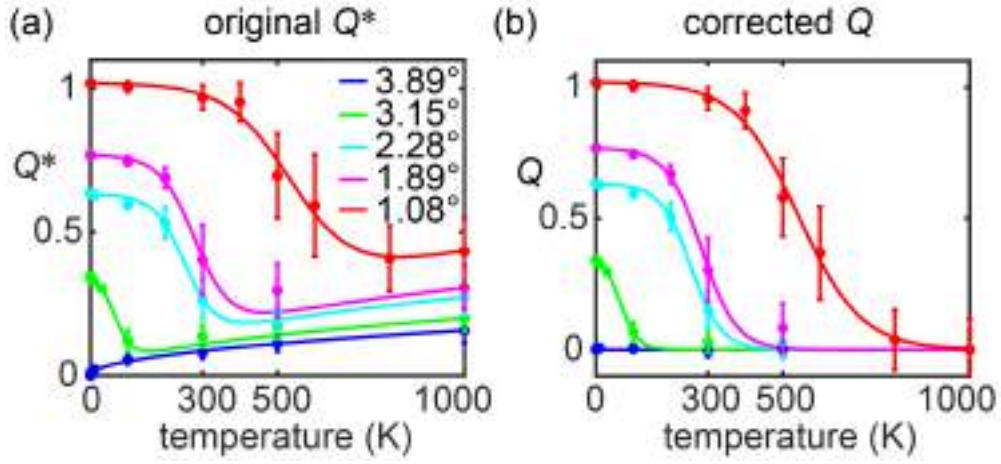


FIG. S3: (a) Uncorrected order parameter Q^* for different twist angles. The solid line fit is based on a smoothed Heaviside step function (see text). (b) Corrected order parameter Q (thermal background term subtracted). Parameters used here are $N_x = 3$ and $N_y = 1$.

TABLE S1: Estimated unbuckling temperature T_c and finite size fitting parameter T_f for different twist angles.

twist angle θ ($^\circ$)	1.08	1.89	2.28	3.15	3.89
T_c (K)	541	283	254	65.2	0
T_f (K)	83.8	46.9	40.8	24.0	\

IV. UNBUCKLING TEMPERATURE T_c

With the corrected parameter Q , we can estimate the unbuckling temperature T_c . First, it is generally seen that flexural fluctuations, large already at relatively low temperatures, play very little role and the buckling order parameter Q remains close to Q_0 so long as ($T \ll T_c$). At very high temperatures ($T \gg T_c$) conversely, buckling disappears, and $Q = 0$.

The actual transition between the two regimes, ideally sharp at infinite size, (and presumably with order parameter exponent β either 1/8 if Ising, or 1/9 if 3-state Potts) is artificially smoothed into a soft crossover at our small simulation sizes. A size-smoothed Heaviside step function (approximated as a sigmoid) $S(T - T_c)$ is used to crudely account for this behaviour $Q(T) = Q_t S$, where $Q_t = Q_t(T)$ is the (unknown) true order parameter and $S(T - T_c) = \{1 + \exp[(T - T_c)/T_f]\}^{-1}$, with T_f a parameter characterizing the finite-size smoothing. We found that a good fit of the finite size crossover of Fig. S3b can already be obtained by crudely setting $Q_t(T) = Q_0$ (a choice compatible with a first order transition, but of course not proving it), with the transition temperature T_c as the only parameter. Values of T_c for different twist angles are shown, along with the fitting parameter T_f in Table S1.

As shown in Fig. S3b, fits based on this crude trial (solid lines) agree well with the MD simulations (color-matched points). The unbuckling transition temperature T_c so obtained for magic twist TBG (see Fig. S3b and Table S1) is as high as ≈ 500 K – one should by all means be able to observe this buckled structure at room temperature.

V. ZIGZAG MODEL DETAILS

Here we give details about the zigzag model and discuss its applicability to understand the bending stiffness of TBG and the scaling exponents.

A. Effective bending stiffness

The model is illustrated in Fig. S4a – a simplified zigzag out-of-plane mechanical structure. When this model structure is bent (Fig. S4b), the hinge angles deform. The model (Fig. S4c) consists of flat (AB-like) regions and hinge (AA-like) regions. The total length of the system (along the bending direction) is L_x , where $L_x = N_x l_x$.

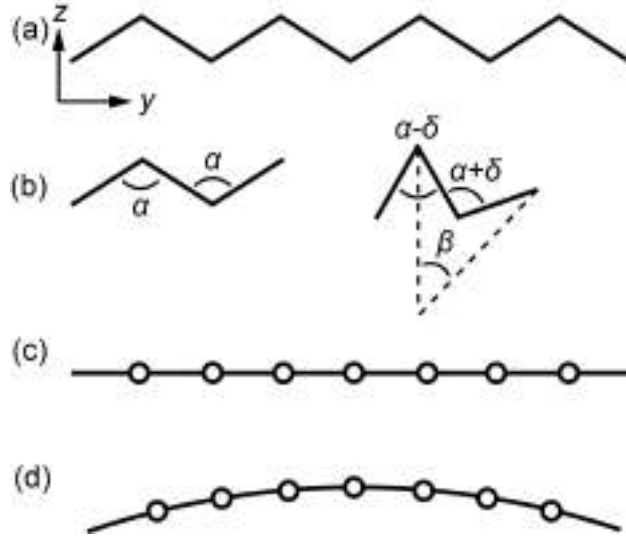


FIG. S4: Schematic diagram of the hinge model for the buckling structure.

When the model is bent (Fig. S4d), both hinge and the flat regions undergo the bending deformations, the relative free energy increase of the system is

$$F(\Phi) = \frac{D_f K l_x l_y \Phi^2}{2L_x (2D_f l_y + K l_x)} \quad (\text{S5})$$

where D_f is the bending stiffness of the flat region, K is the angular stiffness of hinges, and Φ is the total bending angle. Compared to the bending energy expressed by the effective bending stiffness of the whole system,

$$E = \frac{D_{\text{eff}} l_y \Phi^2}{2L_x} \quad (\text{S6})$$

one gets the effective bending stiffness for the zigzag buckled model:

$$D_{\text{eff}} = \frac{D_f}{1 + 2D_f/\sqrt{3}K} \quad (\text{S7})$$

where we used $l_x = \sqrt{3}\lambda$ and $l_y = \lambda$. The bending stiffness of the flat region D_f is approximated by:

$$D_f = \begin{cases} D_B \approx 100 \text{ eV}, & (\theta < \theta_c) \\ 2D_0 = 2.88 \text{ eV}, & (\theta > \theta_c) \end{cases} \quad (\text{S8})$$

To estimate the bending stiffness from Eq. (S7), the only undetermined parameter is the hinge stiffness K .

B. Hinge stiffness K

Here we provide simulation details that reveal the magnitude and θ -dependence of the hinge stiffness K . Since the stiffness is proportional to the square of buckling frequency, $K \propto \rho_{2D}l^4\omega_{\pm}^2$, one can firstly extract the θ -dependence of the buckling frequency $\omega(\theta)$.

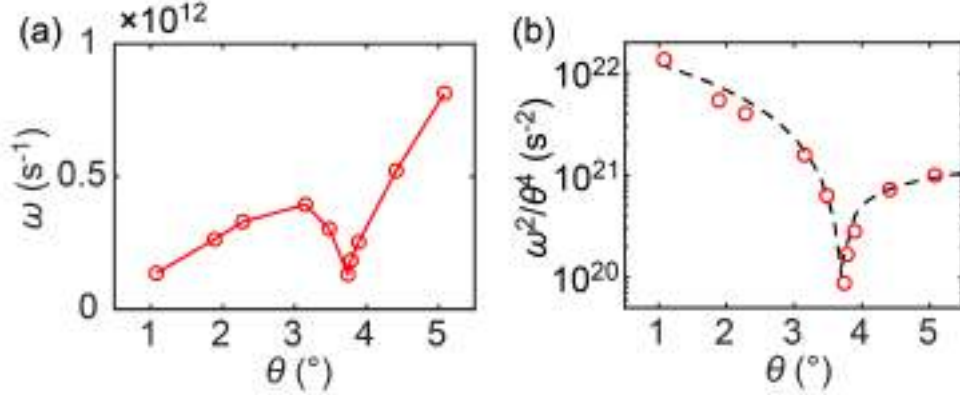


FIG. S5: (a) Dependence of the buckling frequency ω upon the twist angle θ . (b) Twist angle dependence of the “normalized” buckling frequency, ω^2/θ^4 (red). The dashed lines are power-law fits given by Eq. (S10).

For $\theta < \theta_c$, the buckling frequency can be extracted by small oscillations around equilibrium once the dynamics is started with $Q_0 + \delta Q$ (with $\delta Q/Q_0 \ll 1$). For $\theta > \theta_c$, where the static buckling vanishes ($Q_0 = 0$), the buckling frequency can still be extracted by injecting a small initial amplitude buckling distortion ($\Delta z_{\text{max}} \ll d_0$) and then tracking the damped oscillations as the TBG evolves towards its unbuckled ground state. Note that the

frequency of the soft mode so injected, whose eigenvector has the wavelength of the (2×1) buckling unit cell (i.e., 2 moirés), does not change as N_{moire} increases. As shown in Fig. S5, the buckling frequency $\omega \rightarrow 0$ as $\theta \rightarrow \theta_c$.

The stiffness K can be expressed as

$$K = c \frac{\rho_{2D}}{16\pi^4} (\sqrt{3}a_{\text{Gr}})^4 \frac{\omega^2}{\theta^4} \left(\frac{180}{\pi} \right)^4 \quad (\text{S9})$$

where c is a prefactor of $O(1)$. The last term is introduced so as to express the twist angle θ in degrees rather than radians. The twist dependence of K clearly reflects that of ω^2/θ^4 . By fitting the simulation results (red circles in Fig. S5b), we get:

$$\frac{\omega^2}{\theta^4} = \begin{cases} 3.24 \times 10^{21} (\theta_c - \theta)^{1.30} \text{ s}^{-2}, & (\theta < \theta_c) \\ 8.58 \times 10^{20} (\theta - \theta_c)^{0.37} \text{ s}^{-2}, & (\theta > \theta_c) \end{cases} \quad (\text{S10})$$

With the twist angle dependence of K thus obtained, the predictions of the zigzag model are: (a) For $\theta \rightarrow \theta_c$, the buckling structure is infinitely soft ($K \rightarrow 0$), and it follows that $D_{\text{eff}} \rightarrow 0$. This agrees with the bending stiffness collapse discovered in bending simulations. (b) On the low θ side $\theta < \theta_c$, with $D_f \approx 100 \text{ eV} \gg K$, the effective bending stiffness $D_{\text{eff}} \sim K$. According to Fig. S5b, K (and thus D_{eff}) increases as the twist angle decreases from θ_c , with an exponent $\gamma_- = 1.30$ – close to the bending simulation exponent $\epsilon_- = 1.44$. (c) On the large θ side $\theta > \theta_c$ with $D_f \approx 2D_0 = 2.88 \text{ eV} \ll K$, the effective bending stiffness $D_{\text{eff}} \sim 2D_0$. According to Fig. S7b, D_{eff} decreases as the twist angle decreases approaching θ_c , with an exponent $\epsilon_+ = 0.2$ – not too far from the soft mode exponent $\gamma_+ = 0.3(7)$.

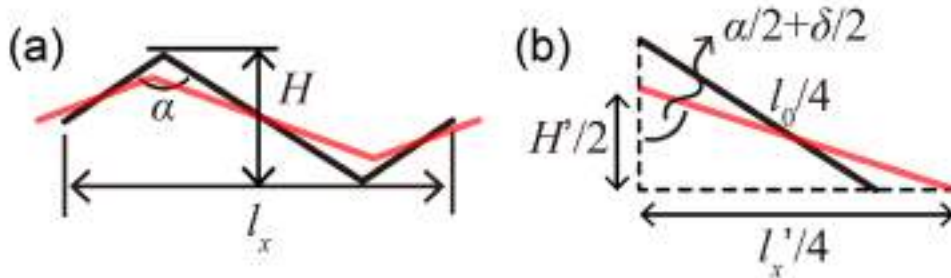


FIG. S6: (a) Model for elongation zigzag model, where black and red represent the equilibrium and elongated structure. (b) The quarter part of the model.

To fully address the magnitude of K and compare theory and bending simulations, we need to specify the value of parameter c in Eq. S9. To achieve this, we performed ad-

ditional “elongation” simulations. The elongation simulations are constructed with quasi-static protocol – stretching the size of the simulation box along x direction by $\delta l_x = 0.002 \text{ \AA}$ ($\delta l_x \ll l_x$) in each step and performing the structural optimization. The total elongation is $\Delta l_x = \delta l_x \times N_{\text{step}}$, where N_{step} is the number of stretching steps.

The elongation simulation is sketched in Fig. S6, where l_0 is the size of the (2×1) unbuckled structure (along x), l_x and H are the size and corrugation height of the buckled structure, l'_x and H' are the size and corrugation height during the elongation simulation, Δl_x is the elongation, α is the equilibrium angle of each hinge and δ is the increase of the angle. These parameters are related by

$$\begin{aligned}
 H &= l_0/2 \cos(\alpha/2) \\
 l_x &= l_0 \sin(\alpha/2) \\
 H' &= l_0/2 \cos(\alpha/2 + \delta/2) \\
 l'_x &= l_0 \sin(\alpha/2 + \delta/2) \\
 \Delta l_x &= l'_x - l_x
 \end{aligned} \tag{S11}$$

where $\delta \ll \alpha$ and $\alpha + \delta \leq \pi$. From the elongation simulation, one can extract the tensile stiffness (of the buckled structure) A by fitting the energy increase $\Delta E = \frac{1}{2}A\Delta l_x^2$. Considering that the flat region can be regarded as a rigid body, this ΔE is mainly contributed by the increase of the hinge energy $2 \times \frac{1}{2}K\delta^2$, where the prefactor 2 represents two hinges in a (2×1) moiré unitcell. Substituting the relationship $\delta = \Delta l_x/H$ (which can be derived from Eqs. S11 above) into ΔE and equating them, we finally have the hinge stiffness $K = AH^2/2$. The value of A and K from elongation simulations is listed in Table S2. Comparing the value of K in Table S2 with Eq. S9 and S10, we find that $c \approx 1.75$ gives the best match.

TABLE S2: hinge stiffness of several buckled structures.

twist angle θ ($^\circ$)	2.87	3.15	3.48	3.67
A (N/m)	39.12	41.36	47.66	46.13
H (\AA)	2.958	2.272	1.323	0.6135
K (eV)	10.69	6.67	2.61	0.543

VI. BENDING STIFFNESS AT FINITE TEMPERATURE

At finite temperature, flexural fluctuations immediately grow for our very large supercells, where the out-of-plane deformation of the buckling structure is heavily influenced by the thermal noise. Thus, it is difficult to use either the oscillation frequency method or the energy method to extract bending stiffness. We introduced an alternative method that is more effective at higher temperatures ($T \gtrsim T_c$), where all soft buckling phonon modes are largely populated. Since we wish to extract the flexural phonon dispersion (thus the bending stiffness) from MD simulations, we turn to the power spectral density (PSD) method [12]. The PSD method projects the velocity of each carbon atom to graphene phonon modes:

$$P(\vec{k}, \omega) = \frac{1}{4\pi\tau N} \sum_{\alpha} \sum_l m_l \left| \int_0^{\tau} \sum_{n=1}^N \dot{u}_{l\alpha\vec{R}_n}(t) \exp(i\vec{k} \cdot \vec{R}_n - i\omega t) dt \right|^2 \quad (\text{S12})$$

where τ is the simulation time, N is the total number of the primitive cell, $u_{l\alpha\vec{R}_n}(t)$ is the displacement of carbon atom in the α direction ($\alpha = x, y, z$) of atom l (with mass m_l) inside primitive cell \vec{R}_n . Here l and n is the atom index inside one graphene primitive cell ($l = 1, 2$) and the index of the primitive cell ($n = 1, 2, \dots, N$) respectively.

In this way, the distribution of kinetic energy in each mode can be directly displayed, yielding the dispersion relation of the system. The dispersion relation of the flexural mode $\omega \propto k^2$, and the bending stiffness of the system can be obtained by fitting the prefactor

$$D = \frac{\rho_{2D}\omega^2}{k^4} \quad (\text{S13})$$

where ρ_{2D} is the mass of the bilayer per unit area. Several typical results of $\log_{10}[P(k, \omega)]$ (maxima used to identify the flexural mode) and the corresponding fit extracting the effective bending stiffness D are shown in Fig. S7.

For low temperature $T \ll T_c$, the buckling order parameter $Q \approx Q_0$, thus we expect that the bending stiffness of the system is dominated by the ‘‘hinge’’ stiffness K , and the value of D should be similar to $T = 0$ cases.

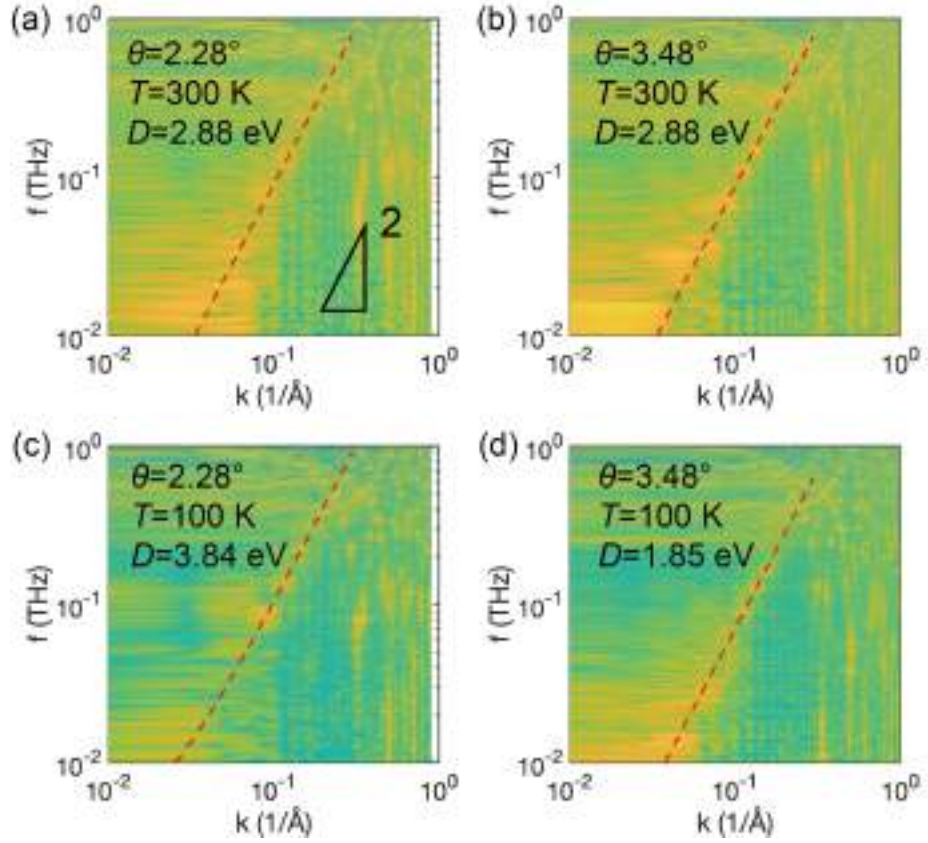


FIG. S7: Power spectral density of the upper layer of a twisted graphene bilayer at finite temperature. (a, b) Room temperature results for $\theta = 2.28^\circ$ and $\theta = 3.48^\circ$. (c, d) $T = 100$ K results for $\theta = 2.28^\circ$ and $\theta = 3.48^\circ$. Here the frequency is $f = \omega/2\pi$. The spectrum for the lower layer is same as the upper layer.

VII. TILT ANGLE

The buckled (2×1) bilayer normal locally deviates from \hat{z} , owing to the tilt of flat AB and BA regions.

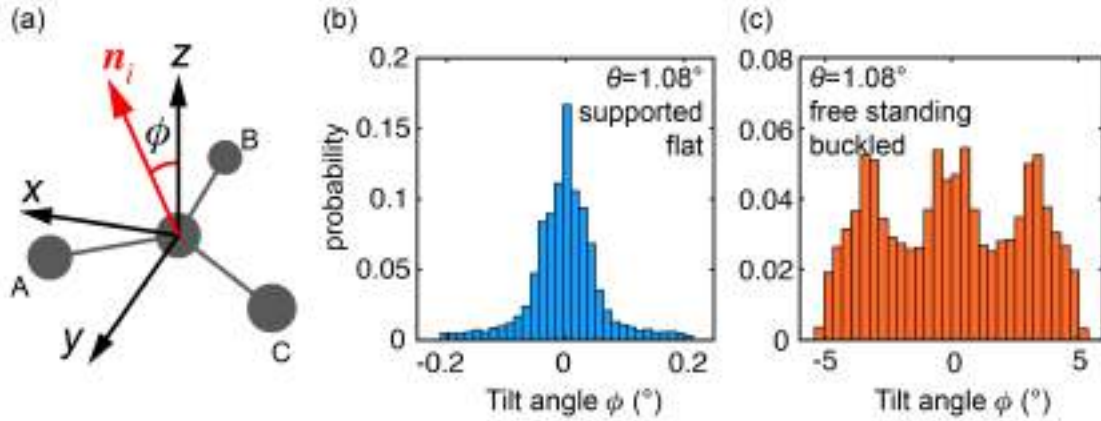


FIG. S8: (a) Schematic diagram of the local normal \vec{n}_i and the tilt angle ϕ . Probability distribution of ϕ for the magic angle (b) un buckled and (c) buckled structures.

The tilt angle of i -th atom is defined as:

$$\phi_i = \arcsin\left(\frac{\vec{n}_i \cdot \vec{e}_x}{|\vec{n}_i|}\right) \quad (\text{S14})$$

where \vec{n}_i is the local normal vector (Fig S8a, with A, B and C the nearest neighbors of the central carbon atom). Take $\theta_m = 1.08^\circ$ TBG ($T = 0$) as an example, the distribution of ϕ for supported-flat structure and freestanding-buckling structure are shown in Fig. S8b and c respectively. The maximum and the most probable tilt angles of the buckling structure ($\phi_{\max} \approx 5^\circ$ and $\phi_{\text{peak}} \approx 3^\circ$) are more than an order of magnitude larger than the flat structure's ($\phi_{\max} \approx 0.2^\circ$). In addition, the distribution is also different, single peak for the flat structure, split peaks for buckled structure. Experiments detecting the tilt angle of the surface should easily capture the buckled structure when present.

VIII. $\sqrt{3} \times \sqrt{3}$ BUCKLING

Besides the 2×1 buckling described in main text, energy minimization in larger simulation cells and zero planar stress found an alternative $\sqrt{3} \times \sqrt{3}$ buckled structure at only slightly higher total energy.

To convey a feeling for the relative difference, at $\theta = 3.15$ degrees for example, the total energy per atom $E_{\text{tot}}/N_{\text{at}}$ is $E_i^{(\sqrt{3}\times\sqrt{3})} = -7.418088$ eV for $\sqrt{3} \times \sqrt{3}$ buckling, against $E_i^{(2\times 1)} = -7.418090$ eV for 2×1 buckling and $E_i^{\text{flat}} = -7.418076$ eV for no buckling. These differences are not as imperceptible as they seem, once scaled with the large supercell size, and more importantly against the modest buckled-unbuckled entropy difference, resulting for example in an unbuckling temperature estimated to be as large as 65 K for the (2×1) bilayer at this twist angle (Table S1).

It might of course happen that, owing to contingent factors such as imperfect structure caused by ill-fitting of $\sqrt{3} \times \sqrt{3}$ buckling inside the rectangular simulation box, force field inaccuracy, different entropies of the two phases at finite temperature, etc., the balance could be reversed, with $\sqrt{3} \times \sqrt{3}$ slightly more stable than 2×1 . Even then, the former would remain an isolated possibility, whereas most uniaxial perturbations, intentional or accidental, would generally stabilize the latter. For this additional reason we concentrated on the 2×1 buckling.

The detailed geometry of the two buckled structures at the same twist angle is shown in Fig. S9. In the $\sqrt{3} \times \sqrt{3}$ distortion, where C_{3z} symmetry is fully restored, the up-down symmetry is broken, with a triangular AA1 superlattice strongly raised out of plane, and a complementary honeycomb superlattice comprising two weakly lowered AA2 and AA3 per cell (Fig. S9b). Further below we will show in Fig. S13, for completeness, the 12 electronic bands of the $\sqrt{3} \times \sqrt{3}$ buckled structure, of course different from those of 2×1 .

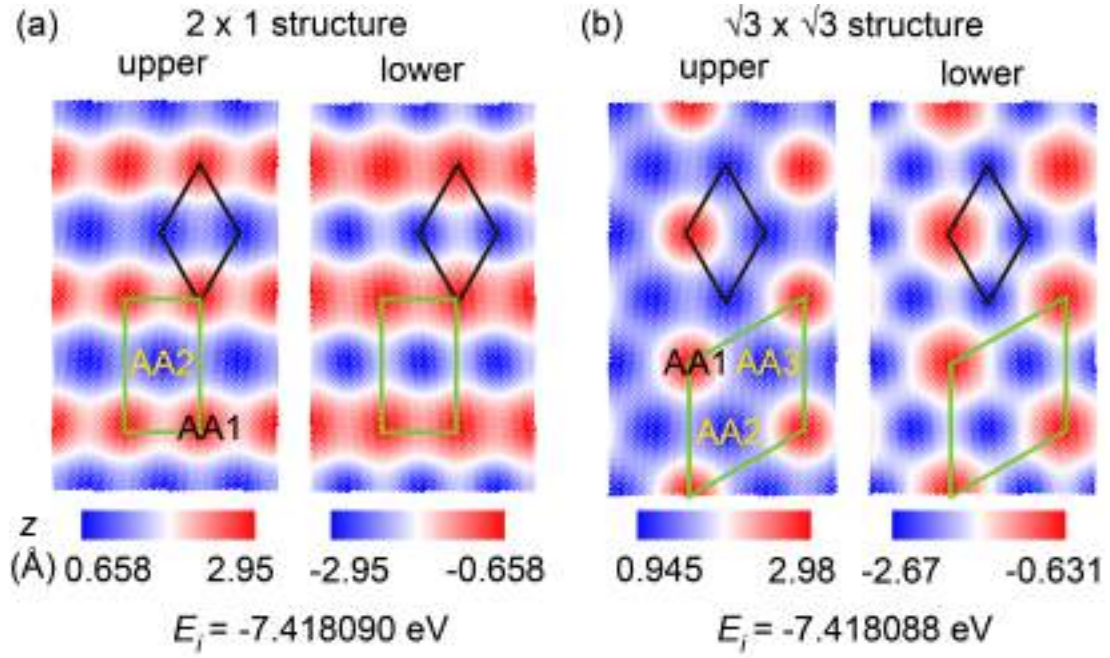


FIG. S9: The 2×1 and $\sqrt{3} \times \sqrt{3}$ buckling structures for $\theta = 3.15^\circ$. The moiré primitive cell is highlighted by the black rhombus; the corresponding 2-moiré and 3-moiré cells and simulation boxes are shown with green rectangle/rhombus. The corrugation height and the potential energy E_i per atom are listed at the bottom.

IX. ELECTRONIC STRUCTURE

A. Hamiltonian

The tight binding Hamiltonian is $H = \sum_{i,j} -t_{ij}|i\rangle\langle j| + \text{H.c.}$, where t_{ij} is the transfer integral between sites i and j . Let $\mathbf{R}_j - \mathbf{R}_i = (l, m, n)d$ be the distance vector between two orbital, and $\hat{\mathbf{n}}_i = (n_i^l, n_i^m, n_i^n)$ be the direction of orbital \mathbf{p}_i , then using the Slater-Koster[13] formula

$$t_{ij} = \sum_k^{l,m,n} n_i^k \left[n_j^k (k^2 V_{pp\sigma} + (1 - k^2) V_{pp\pi}) + \sum_{g \neq k}^{l,m,n} n_j^g g k (V_{pp\sigma} - V_{pp\pi}) \right] \quad (\text{S15})$$

where the out-of-plane (σ) and in-plane (π) hoppings are

$$V_{pp\sigma} = V_{pp\sigma}^0 \exp\left(-\frac{d - d_0}{r_0}\right), \quad V_{pp\pi} = V_{pp\pi}^0 \exp\left(-\frac{d - a_0}{r_0}\right) \quad (\text{S16})$$

where $V_{pp\pi}^0 = -2.7$ eV and $V_{pp\sigma}^0 = 0.48$ eV are chosen to reproduce ab-initio curves in AA and AB stacked bilayer graphene similar to previous works [14], $d_0 = 3.344$ Å is the minimum distance between the two layers in the AB stacked region, $a_0 = a_{\text{Gr}}/\sqrt{3} \approx 1.42$ Å is the carbon-carbon bond length in the relaxed structure, and r_0 is the decay rate $\approx 0.184a_{\text{Gr}}$.

In the particular case of unbuckled, flat bilayer $\hat{\mathbf{n}}_i \approx \hat{\mathbf{n}}_j \approx (0, 0, 1)$ therefore Eq. (S15) reduces to $t_{ij} = n^2 V_{pp\sigma} + (1 - n^2) V_{pp\pi}$

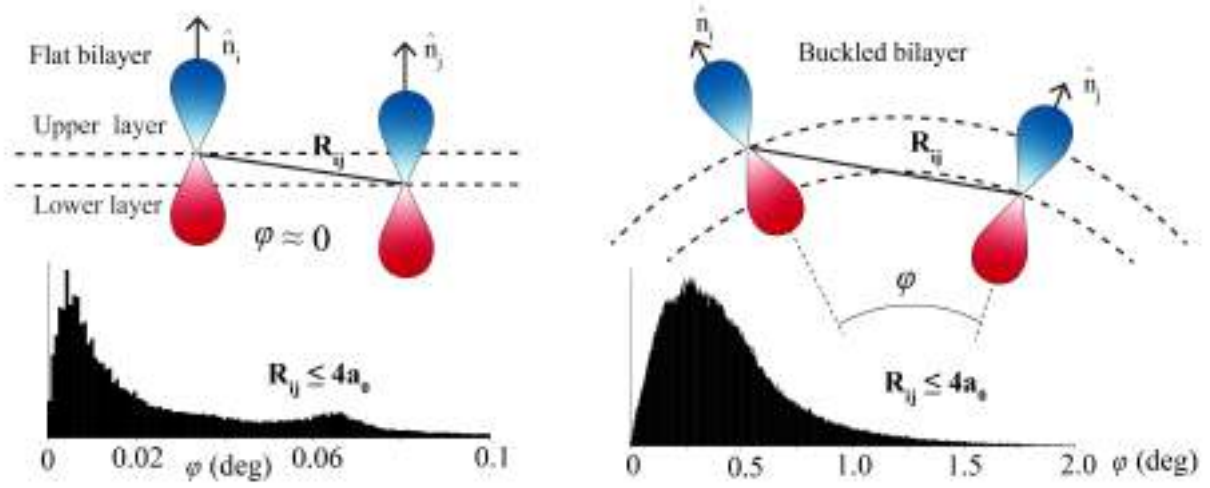


FIG. S10: Unlike the flat bilayer (left panel), where local normals deviate very little from vertical, the deviation is much larger in the buckled state (right panel). The distribution of the included angle φ between two local normals \hat{n}_i and \hat{n}_j (within the cutoff range $R_{ij} \leq 4a_0$) is shown in the lower panel. The maximum φ for the buckled structure is more than one order of magnitude larger than the unbuckled structure – similar to what we discovered for the tilt angle ϕ .

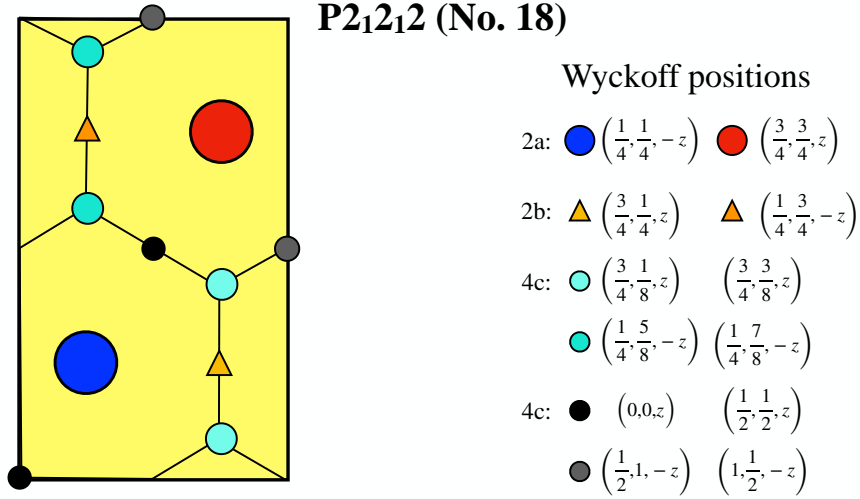


FIG. S11: Unit cell and Wyckoff positions in the space group $P2_12_12$. Note that the 2a positions corresponds to the AA stacked regions, whereas the 4c positions in cyan to the Bernal stacked ones.

B. Space group

The unit cell of the (2×1) buckled TBG together with the Wyckoff positions in the $P2_12_12$ space group (no. 18) is shown in Fig. S11. We note that the 2a Wyckoff positions (red/blue circles) correspond to the AA stacked regions and the 4c positions (cyan circles) to the Bernal stacked ones. The 3f positions in the P622 space group (no. 177) of the unbuckled TBG split into the 2b and 4c (yellow triangles and black circles, respectively) in the space group of the buckled TBG. The non-symmorphic symmetry connects the two 2a positions, the two 2b ones, as well as the two sets with opposite z coordinates within the 4c Wyckoff positions (light cyan \rightarrow dark cyan, as well as dark gray \rightarrow black).

C. Symmetries

In Table S3 we report the character table of the irreducible representations (irreps) at the high-symmetry points in space group $P2_12_12$, where σ_a , $a = 1, 2, 3$, are Pauli matrices that act in the space of two-dimensional irreps. The eigenstates of the Hamiltonian that we

	E	C_{2z}	C_{2y}	C_{2x}
Γ_1	1	1	1	1
Γ_2	1	1	-1	-1
Γ_3	1	-1	-1	1
Γ_4	1	-1	1	-1

	E	C_{2z}	C_{2y}	C_{2x}
W_1	1	1	i	i
W_2	1	1	$-i$	$-i$
W_3	1	-1	$-i$	i
W_4	1	-1	i	$-i$

	E	C_{2z}	C_{2y}	C_{2x}
$X_1(2)$	1	σ_3	σ_1	$-i\sigma_2$
$Y_1(2)$	1	σ_3	$-i\sigma_2$	σ_1

TABLE S3: Character table and irreducible representations at the high-symmetry points in space group $P2_12_12$.

calculate do transform as one of the irreps at the corresponding high-symmetry points, see Fig. S12(a), supporting our identification of the space group.

In Fig. S12(b) we also show the flat bands of the *unbuckled* bilayer folded into the reduced BZ of the 2×1 buckled one, labelling the Bloch waves at the high symmetry point with the irreps of the $P2_12_12$ space group, (despite the correct space group in this case being $P622$, no. 177). We remark that, proceeding as outlined in the main text, in this unbuckled case too we can identify the flat bands corresponding to a given valley, magnetic space group $P2_12'_12'$, no. 18.19. Like in the buckled case, these single-valley bands can be generated by the atomic limit of Wannier orbitals centered at the Wyckoff positions $4c$ of space group $P2_12'_12'$. Yet, this was not possible in the original unfolded band structure [14, 15]: and that was a clear symptom of non-trivial topological character of the unbuckled bilayer – *a property that should be independent of the arbitrary choice of unit cell and folding procedure*. For that reason, despite our inability to prove it, we conjecture that the bands of the buckled twisted bilayer graphene are likely to be topological as well.

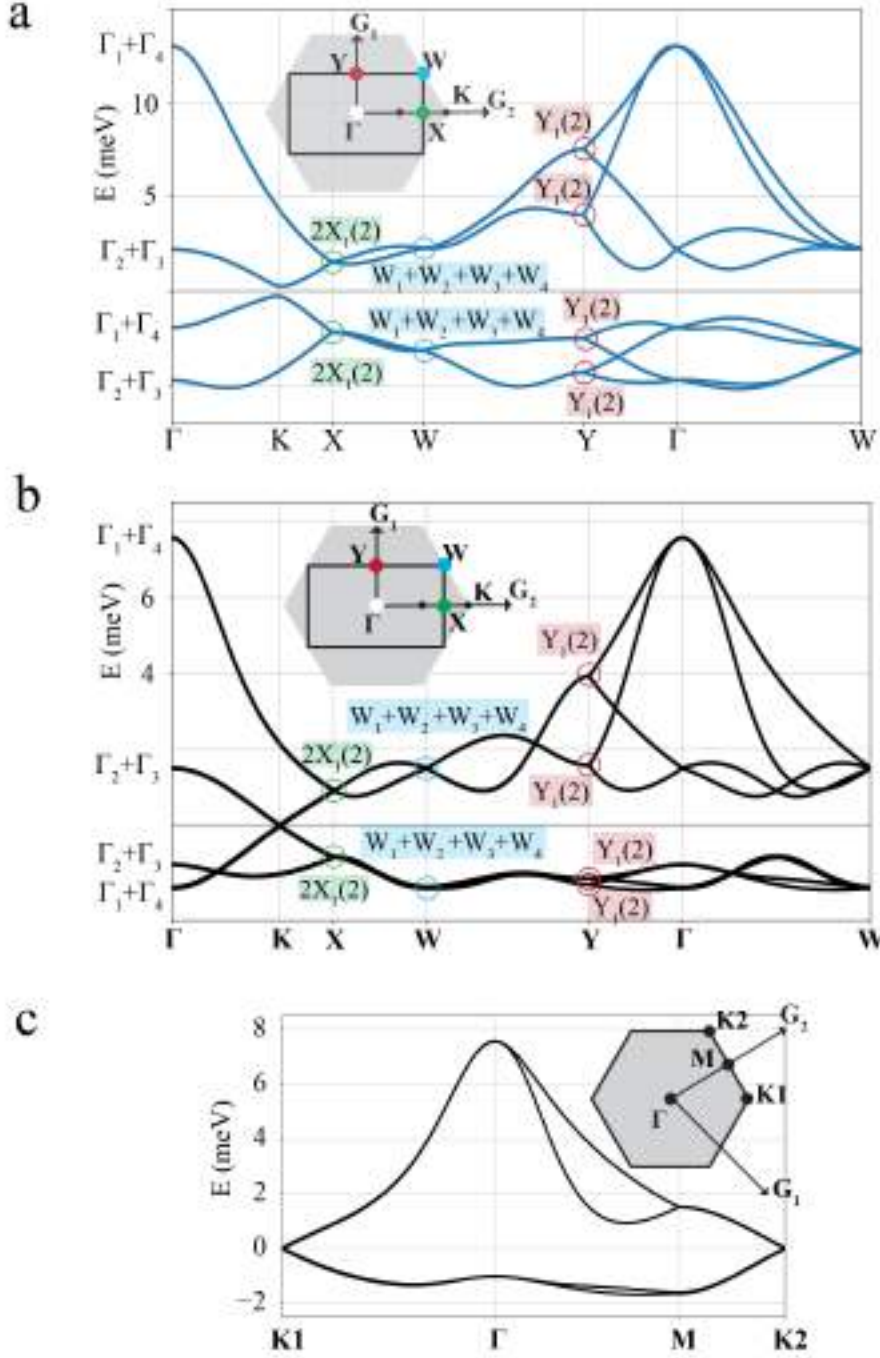


FIG. S12: Flat bands for the bilayer graphene with twist angle $\theta = 1.08^\circ$. Top panel (a): flat bands of the (2×1) buckled bilayer; middle panel (b): flat bands of the unbuckled bilayer folded into the (2×1) reduced Brillouin Zone. The additional crossings of the unbuckled bands arise because the actual space group is P622 (no. 177). Bottom panel (c): flat bands of the unbuckled bilayer in their natural (1×1) Brillouin Zone.

For completeness, the flat bands of the $\sqrt{3} \times \sqrt{3}$ buckled structure are shown in Fig. S13.

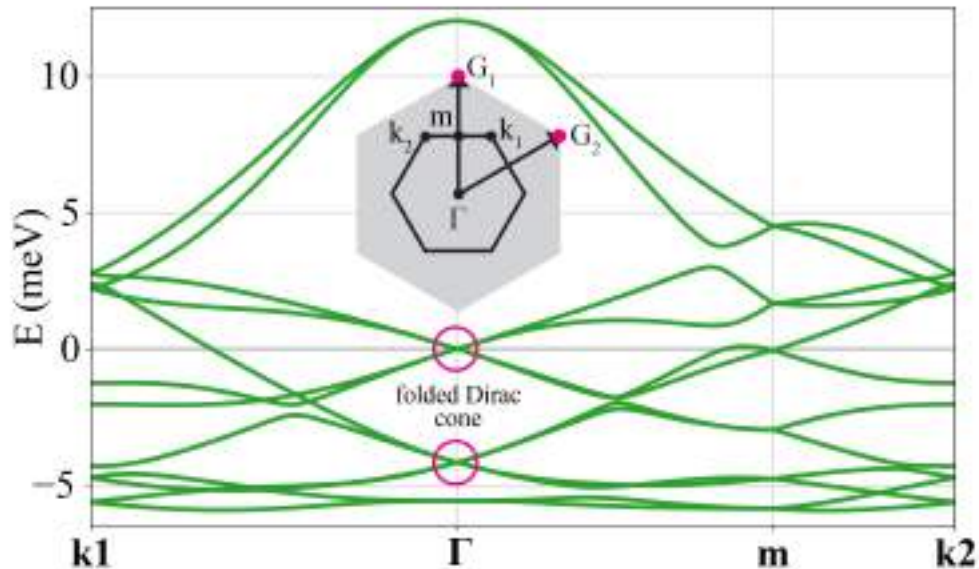


FIG. S13: Flat bands for the $\sqrt{3} \times \sqrt{3}$ buckled bilayer with twist angle $\theta = 1.08^\circ$. Note the double folding of the Dirac cone at Γ . Beware of the 90 degrees rotated convention compared with that of (2×1) bands of other figures.

D. LTB+Symm code

For this work, we developed the LTB+Symm code (Large-scale Tight Binding and Symmetry validation). LTB+Symm is a useful tool to perform tight binding calculations with non-planar geometries in large-sized unit cells, thanks to its MPI implementation. LTB+Symm is an object-oriented, open-source Python3 software; its streamlined, geometry-agnostic implementation allows to easily implement *ad-hoc* solutions to capture system-specific features. A novel aspect of LTB+Symm is its ability to validate and label irreps at high-symmetry points of the Brillouin zone (once the right space group is defined). LTB+Symm is publicly accessible on GitHub: <https://github.com/khsrali/LTB-Symm>.

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